# МIHICTEPCTBO ОСВITИ I НАУКИ УКРАЇНИ ТЕРНОПІЛЬСЬКИЙ НАЦІОНАЛЬНИЙ ТЕХНІЧНИЙ УНІВЕРСИТЕТ ІМЕНІ ІВАНА ПУЛЮЯ 

## КАФЕДРА ТЕХНОЛОГІЇ МАШИНОБУДУВАННЯ

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## МЕТОДИЧНІ ВКАЗІВКИ

# до виконання курсової роботи 3 дисципліни «ТЕХНОЛОГІЯ ОБРОБКИ ДЕТАЛЕЙ ТИСКОМ» 

для здобувачів вищої освіти<br>3 числа іноземних громадян за спеціальністю 131 - «Прикладна механіка»

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Методичний посібник розглянуто й затверджено на засіданні кафедри технології машинобудування
Тернопільського національного технічного університету імені Івана Пулюя.
Протокол № 7 від 30 березня 2018 р.

Схвалено та рекомендовано до друку на засіданні методичної ради факультету інженерії машин, споруд та технологій
Тернопільського національного технічного університету імені Івана Пулюя.
Протокол № 1 від 31 серпня 2018 р.

Методичні вказівки до виконання курсового проекту з дисципліни M54 «Технологія обробки деталей тиском» для здобувачів вищої освіти з числа іноземних громадян за спеціальністю 131 - «Прикладна механіка» / Укладачі : Радик Д.Л., Паньків М.Р., Паньків В.Р., Сіправська М.Д. Тернопіль : Тернопільський національний технічний університет імені Івана Пулюя, 2018. - 44 с.

УДК 621.77

Відповідальна за випуск асистент Сіправська М. Д.

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# METHODICAL INSTRUCTIONS <br> to coursework from the discipline <br> ${ }^{6}$ TECHNOLOGY OF WORKPIECE PRESSURE SHAPING" 

for Mechanical Engineering major students

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Examined and approved at the methodological seminar of the manufacturing engineering department
Ternopil national technical university named after Ivan Pul'uj. Protocol № 7 від 30 March 2018 r.

Recommended for publication by methodical commission of the faculty of engineering machinery, facilities and technologies,
Ternopil national technical university named after Ivan Pul'uj. Protocol № 1 від 31 August 2018 r.

Methodical instructions to coursework from the discipline "Technology of M54 workpiece pressure shaping" for Mechanical Engineering major students/ Authors : Raduk. D.L., Pankiv M.R., Pankiv V.R., Sipravska M.D. - Ternopil : Ternopil national technical university named after Ivan Pul'uj, 2018. - 44 c .

УДК 621.77

Responsible for the publication Sipravska M. D.

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## INTRODUCTION

In the general complex of technology of mechanical engineering acquired great significance such branch as the treatment of metals by pressure, which includes such operations as manual and machine forging, sheet and bulk stamping, pressing, rolling, drawing, etc.

The most progressive and widespread method of processing metals by pressure in modern mechanical engineering is stamping, which consists in plastic deformation of metal with the help of dies. Punching can be carried out both from a bulk billet by rolling, drawn or pressed rod (or thick band), and from sheet blanks - sheet material. The first process is called bulk stamping, and the second is sheet stamping.

In modern engineering, a significant part of products manufactured by punching. Sheet punching is a progressive type of material processing by pressure and using the sheet material as the initial blank (sheets, stripes, tapes), as well as technological transitions (punching, bending, pulling) allows us to produce a wide range of different types of flat and bulk products.

Sheet punching has a fairly wide application in all branches of machine and instrument making, metalworking, radio and electrical engineering. Preferably it is used in mass and batch production, as well as in small-scale, using universal punches.

The widespread use of sheet stamping techniques is provided by the following main advantages: high productivity and low cost of stamped parts; the possibility of manufacturing products with a minimum material capacity that can't be provided by other methods of processing; high accuracy and quality of the surface of pressed parts, ensuring their interchangeability; relatively simple mechanization and automation of technological processes of sheet stamping; the possibility of obtaining products of complex form and configuration and high coefficient of material use; adaptability to scale of production.

Depending on the thickness of the sheet blank, the stamping is conventionally divided into thin-walled ( $\mathrm{s}<4 \mathrm{~mm}$ ) and thick-sheeted ( $s>4 \mathrm{~mm}$ ), preforms with a thickness of $15-20 \mathrm{~mm}$ are generally stamped in hot form.

The range of sizes of stamped parts is quite large - in dimensions from a few millimeters to $6-7 \mathrm{~m}$; in the thickness from a tenth of a millimeter to 100 mm or more. The precision of manufacturing parts is achieved with ordinary stamping grades 4-5, and with additional cleansing and calibration of grade 3-2 accuracy.

Typical content
Of explanatory note to the coursework
From subject

## "TECHNOLOGY OF WORKPIECE PRESSURE SHAPING"

On the topic:
"Development of manufacture process of stamping a part ..."

Introduction

1. ANALYSIS OF PART DESIGN FEATURES.
1.1 Part function and characteristic.
1.2 Analysis of technical requirement and mechanical properties of part material.
1.3 Analysis of part design manufacturability.
2. BLANK CHOOSING.
2.1 Development of sheet metal pattern cutting scheme.
2.2 Sheet products choosing.
2.3 Utilization rate determining.
3. DETERMINING OF STAMPING ENERGY-POWER PARAMETERS.
3.1 Calculation of cutting force (punching).
3.2 Part pushing (removal) force determining.
3.3 Spent working on punching and press power calculation.
4. STAMP CALCULATION AND DESIGNING.
4.1 Stamp pressure center determining.
4.2 Stamp closed height determining.
4.3 Punch and die gaps selecting and executive (operative) sizes determination.
4.4 Stamp stability determination.
4.5 Stamp units designing (punch, die, stops and others).
4.6 Stamp design and operative principles describe.
5. CHOOSING OF EQUIPMENT AND ITS TECHNICAL CHARACTERISTICS.

## CONCLUSIONS

REFERENCES
Additions:
Addition A - specifications;
Addition B - technical documents.

## 1. GENERAL PROVISIONS

The course work is carried out in order to consolidate the knowledge gained by students during theoretical study of the discipline "Technology of processing details by pressure" and acquiring practical skills during the technological preparation of production for sheet stamping, in particular in the selection of the workpiece and the determination of the coefficient of material use, the calculations of the power-supply parameters of the stamping process, the design of the stamp equipment, the choice of equipment and the development of stamping process .

When performing the course work, the following sections shall be developed:

- analysis of structural features of the part;
- choice of type of workpiece;
- determination of power-strength parameters of the stamping process;
- calculation and design of the stamp;
- equipment selection and its technical characteristics;
- development of the process of stamping.

The initial data for course work is an individual task for course work, drawing of the details, material and type of sheet metal.

Individual task students receive during practical classes at the beginning of the semester.

While performing the course work all explanations to the developed points of questions and calculations should be explained specifically with consistent, detailed explanations and references to literary sources and standards from which specific data are derived.

Coursework consists of a settlement and explanatory note and a graphic part.
The settlement and explanatory note for the course work should contain $25-30$ printed sheets of the A4 format and have the following structural elements: title page, tasks for term paper (completed task form and work drawing details), calculation part (according to the points and sub-clauses of the typical content of the settlement- explanatory note to course work), conclusion, list of references, annexes (specification and technological documentation).

The volume of the graphic part in the course work should be $1-1,5$ sheets of A1 format and contain a drawing of the stamp as well as working drawings of the matrix and punch, which simultaneously serve for cutting (punching).

## 2. RECOMMENDATIONS FOR CALCULATION AND EXPLANATORY NOTES

In the introduction to the course work must determine what is a sheet stamping, its advantages over other types of metal processing (technical, economic, constructive, technological). It is also necessary to indicate the areas of sheet stamping and its place and peculiarities in the general structure of machinery engineering. Specify in what types of production the most widely used sheet punching and why? For which details, this method of manufacture is the most acceptable.

## 1. Analysis of the structural features of the part

### 1.1. Purpose and characteristics of the part.

In this question, it is necessary to indicate where this item can be used, what it serves, what its functional purpose and which load can perceive. What type of attachment in the node of the unit of which it can be.

Next it is necessary to characterize the design features of the detail to which class it should be attributed. Describe the geometric details of the part, as well as do a conclusion on the most appropriate method of its manufacture.

Example: Detail "Shield" on the form refers to the flat parts and by the design is a plate, with overall dimensions of $50 \times 100 \mathrm{~mm}$ and a thickness of 3 mm , with an inner hole with a diameter 12 mm . Since the material of the workpiece is a sheet of hot-rolled steel, Art. 3, this part should be made by punching, in particular cutting off by the outer contour and piercing the hole with a stamp.

### 1.2. Analysis of technical requirements and mechanical properties of material

 partsIn the question of the analysis of technical requirements it is necessary to analyze what requirements for accuracy of sizes (qualification of accuracy, accuracy tolerance on the sizes of surfaces) and surface roughness refer to this part. What surfaces can't be processed, and those that are processed in the process of stamping need to analyze whether further mechanical treatment is needed, to give detailed information of the material details.

The mechanical properties of the material of the workpiece, from which the component is manufactured, can be taken from Annex 1 and presented in the form of Table 1:

Table 1 - Mechanical properties of the material of the part

| $\sigma_{\mathrm{T}}\left(\sigma_{02}\right)$, | $\sigma_{\mathrm{B}}$, | $\sigma_{\mathrm{cp}}\left(\tau_{\mathrm{o}}\right)$, | $\delta$ |  | $\Psi$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{kgf} / \mathrm{mm}^{2}$ | $\mathrm{kgf} / \mathrm{mm}^{2}$ | $\mathrm{kgf} / \mathrm{mm}^{2}$ | $\delta_{5}, \%$ | $\delta_{10}, \%$ |  |
|  |  |  |  |  |  |

where $\sigma_{\mathrm{T}}\left(\sigma_{02}\right)$ - yield stress; $\sigma_{\mathrm{B}}-$ tensile strength; $\sigma_{\mathrm{cp}}\left(\tau_{\mathrm{o}}\right)-$ shear resistance; $\delta-$ fineness ratio, in accordance $\delta_{5}-$ at break of a short sample, length equal to five diameters, $\delta_{10}$ - at break of a long sample, the length is equal to ten diameters; $\Psi-$ relative narrowing of the cut.

### 1.3. Analysis of the technological design of the detail

The technological processes of cold stamping can be the most rational only if the technological design or form of the part is created, which ensures its simplest and most economical production. Therefore, the processability of sheet-stamped parts is the most important prerequisite for the progressiveness of technological methods and cost-effectiveness of production.

Under the fabricability of stamped parts, should be understood as a set of properties and structural elements that provide the simplest and economical manufacturing of parts, subject to technical and operational requirements to them.

The operational and technical requirements for sheet stamped parts are mainly as follows:

- full compliance of the design with the purpose and conditions of operation of parts;
- providing the required durability and rigidity of the parts at a minimum metal outlet;
- providing the necessary accuracy and interchangeability;
- compliance with special physical, chemical or technical conditions.

Key indicators of manufacturability of sheet cold forged parts are:

- the smallest material dissipation;
- low labour input of operations;
- absence of subsequent machining;
- the smallest amount of necessary equipment and production space;
- the smallest amount of equipment with simultaneous reduction of costs
and terms of preparation of production;
- increase of production productivity.

The general efficiency indicator of fabricability is the least cost of stamped parts.

Since the value and the ratio of the self-cost of products elements (material, wages, workshop discharges) depend on the seriality of production, then the concept of technology is inextricably linked with the type of production. Technological construction in the conditions of small-scale production may turn out to be nontechnological in mass production and vice versa.

In most cases, the main criterion for the technological design of the component is the most economical material dissipation with the least number of operations and the reduction of labor intensity.

Basic technological requirements for the design of flat parts, obtained by cutting and punching.

1. It is necessary to avoid complicated configurations with narrow and long contour cutouts or very narrow cavities (b> 2 S ), (Fig. 1).


Figure 1 - Examples of parts configuration:
a) with a narrow and long cut; b) with a narrow slot
2. When punching (cutting off) in stamps with integral matrices conjugation in the corners of the inner contour of the parts should be performed with a radius of rounding $r \geq 0,5 \mathrm{~S}$ (Fig. 2).

In composite matrices, the conjugation of the sides perform without rounding (Figure 3)


Figure 2 - An example of the configuration of a part with a pierced inner contour


Figure 3 - Structure of the composite matrix:
$1,2,3,4$ - parts of the matrix; 5 - matrix holder
3. Concussion of the sides of the outer contour should be done with rounding only during cutting out of the part over the entire circuit. In case of possible application of non-waste cutting should on the contrary, to admit and perform conjugation of the parties at right angles.
4. The smallest sizes of breakthrough openings should be selected according to tabular data of Table. 2

Table 2 - The smallest sizes of breakthrough openings

| Material | Normal puncture with a free <br> punch |  |  | Punching in a clamped <br> state with a directional <br> punch |  |
| :--- | :---: | :---: | :---: | :---: | :---: |
|  | Round | Rectangular | Round | Rectangular |  |
| Solid steel | $1,3 S$ | $1,0 S$ | $0,50 S$ | $0,40 S$ |  |
| Soft steel and brass | $1,0 S$ | $0,7 S$ | $0,35 S$ | $0,30 S$ |  |
| Aluminum | $0,8 S$ | $0,5 S$ | $0,30 S$ | $0,28 S$ |  |

5. The smallest distance from the edge of the hole to the rectilinear outer contour must be not less than S for figured round holes and not less than $1,5 \mathrm{~S}$ if the edges of the hole are parallel to the contour of the part (Fig. 4).


Figure 4 - The layout of the arrangement of round and rectilinear holes relative to the external contour of the part, when cutting (punching)
6. The smallest distance between the openings while simultaneously punching them should be in $=(2 \div 3) \mathrm{S}($ Fig. 5)


Figure 5 - An example of the arrangement of the holes on the detail, which at the same time break through

## 2. Choice of workpiece

### 2.1. Development of the material cutting scheme

The cutting out scheme of the material during sheet stamping is called the method of arrangement of cut pieces on the preform (sheet, strip, tape). In stamp production, material savings are largely determined by the cutting, it means the most rational placement of parts on the workpiece. The choice of the method of cutting the material largely depends on the design and dimensions of the stamped part.

Cutting of round parts can be done in one, two and several rows with their parallel and chess placement. Rounded parts with a diameter of more than 150 mm are usually cut in one row, with smaller sizes it is more economical to punch in several rows in a chess placement.

When cutting off rectangular and figured parts, the following types of cutting are used: straight, inclined, facing straight and facing inclined, combined, multi-row, with cut-off jumpers.

In addition, by the method of cutting, cutting out scheme ca be with jumpers and without jumpers. In the latter case, stamping is called low-waste or non-waste.

This point of the course work involves the development of an optimal scheme for cutting sheet into strips or parts to ensure the required number of products. Designing cutting out scheme cutting sheet material is necessary to minimize the amount of material waste. The results of the development presented in the form of graphical schemes of cutting the sheet or strip. Also, calculate the number of strips received from one sheet, the number of parts received from one lane and the total number of parts obtained from the sheet.

For a graphic representation of the cut-out scheme, it is necessary to represent in a scale several details (depending on the stamping method), taking into account the gaps between the part and the workpiece edge (a), between the adjacent parts (a1) and the step $(\mathrm{K})$ when stamping this part.

Table 3.1 - The values of barriers $a$ and $a_{1}$ recommended for parts with round shape for pattern cutting on sheet material


Table 3.2 －The values of barriers $a$ and $a_{l}$ recommended for parts with square shape for pattern cutting on sheet material．

| Material thickness S，mm |  |  |  |  |  | $\begin{aligned} & 1 \\ & \vdots \\ & \vdots \\ & \hline \end{aligned}$ |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Part size $l$ ，mm |  |  |  |  |  |  |  |
|  | Less than 50 |  | 50－100 |  | 100－200 |  | 200－300 |  |
|  | $a_{1}$ | $a$ | $a_{1}$ | $a$ | $a_{1}$ | $a$ | $a_{1}$ | $a$ |
| Less than 0，2 | 2，0 | 2，5 | 2，5 | 3，0 | 3，0 | 3，5 | 3，5 | 4，0 |
| 0，2－0，5 | 1，5 | 1，8 | 1，7 | 2，0 | 2，2 | 2，5 | 2，7 | 3，0 |
| « 0，5 « 1，0 | 1，0 | 1，5 | 1，2 | 1，7 | 1，7 | 2，2 | 2，2 | 2，7 |
| « 1，0 « 1，5 | 1，4 | 1，9 | 1，6 | 2，1 | 2，1 | 2，6 | 2，6 | 3，1 |
| « 1，5《2，0 | 1，7 | 2，2 | 1，9 | 2，4 | 2，5 | 3，0 | 2，9 | 3，4 |
| « 2，0《2，5 | 2，2 | 2，6 | 2，4 | 2，8 | 2，9 | 3，3 | 3，4 | 3，8 |
| « 2，5＜3，0 | 2，5 | 3，0 | 2，7 | 3，2 | 3，2 | 3，7 | 3，7 | 4，2 |
| « 3，0《3，5 | 2，9 | 3，4 | 3，1 | 3，6 | 3，6 | 4，1 | 4，1 | 4，6 |
| « 3，5《4，0 | 3，2 | 3，7 | 3，4 | 3，9 | 3，9 | 4，4 | 4，4 | 4，9 |
| « 4，0 « 5，0 | 3，6 | 4，0 | 3，8 | 4，2 | 4，3 | 4，7 | 4，8 | 5，2 |
| « 5，0《6，0 | 4，0 | 4，5 | 4，5 | 5，0 | 5，0 | 5，5 | 5，5 | 6，0 |
| « 6，0 « 7，0 | 4，5 | 5，0 | 5，0 | 5，5 | 5，5 | 6，0 | 6，0 | 6，5 |
| « 7，0《8，0 | 4，8 | 5，3 | 5，5 | 6，5 | 6，0 | 7，0 | 6，8 | 7，8 |
| « 8,0 « 9，0 | 5，3 | 5，8 | 6，0 | 7，0 | 6，5 | 7，5 | 7，0 | 8，0 |
| « $9,0 \ll 10,0$ | 5，8 | 6，3 | 6，5 | 7，5 | 7，0 | 8，0 | 7，5 | 8，5 |

### 2.2. Choice of sheet metal

When elaborating this subparagraph, determine the type of workpiece based on the task for the course work. The material, according to the type of workpiece, is divided into sheets, stripes, tapes (rolls).

Recommended sizes of sheets: $710 \times 1420,1000 \times 2000,1250 \times 2500$, $1500 \times 3000,2000 \times 5000 \mathrm{~mm}$.

### 2.3. Calculation of material use coefficient

The indicator which characterizes efficiency of pattern cutting is the coefficient of material use $(\eta)$, which is the ratio of the useful area of the part $\left(\mathrm{F}_{\mathrm{o}}\right)$ to the area of the workpiece $\left(\mathrm{F}_{3}\right)$.

For the manufacture of one part when cutting from the piece workpiece (cassette), the coefficient of material use is determined by the formula (1).

$$
\begin{equation*}
\eta_{1}=\frac{F_{0}}{F_{3}} \cdot 100 \% \tag{1}
\end{equation*}
$$

The rationality of using the strip area (tape) is determined by the formula (2)

$$
\begin{equation*}
\eta=\frac{F_{01} \times n}{F_{3}} \cdot 100 \% \tag{2}
\end{equation*}
$$

where $F_{01}$ - area of one part(detail), mm;
$F_{3}=B \times L$ - tape area, mm;
$n$ - number of parts in one tape;
$B, L$ - tape length and width, mm.
The coefficient of the material use of the entire sheet is determined by the formula (3)

$$
\begin{equation*}
\eta_{n}=\frac{N_{\partial} F_{01}}{F_{n}} \cdot 100 \% \tag{3}
\end{equation*}
$$

where $N_{\partial}$ - total number of parts received from the sheet, pcs;
$F_{01}$ - area of one part, $\mathrm{mm}^{2}$;
$F_{\pi}-$ sheet area, $\mathrm{mm}^{2}$.

## 3. Determination of energy-power parameters of stamping process

### 3.1. Calculation of cutting force (punching)

The effort required to overcome the cut resistance of the metal at cutting (punching) in stamps, depends on several factors.

1. Factors related to material, shape and part dimensions: the mechanical properties of the metal and the depth of punch penetration into the metal; the thickness of the sheet material from which part is cut down; shape and size of harvested circuit.
2. Factors related to the stamp design include: the size of the gap between the die and punch, die design (hole shape for output parts or waste).
3. Cutting (punching) terms: deformation speed; lubrication of material and tools; shape of cutting edges of the punch and die, which may be flat, i.e. parallel and oblique - inclined to each other; wear state and the hardness of the blades.

Calculated effort in the process of cutting in stamps with parallel cutting blades when the angle $\varphi=0^{\circ}$, and cutting angle is $\delta=90^{\circ}$, is determined by the formula:

$$
\begin{equation*}
P=F_{o} \sigma_{3 p}=u s \sigma_{3 p} \tag{4}
\end{equation*}
$$

where $-F_{o}=u s-$ cutting area, $\mathrm{mm}^{2}$;
$u$ - contour length (perimeter) of cutted parts or holes that punching, $\mathrm{mm}^{2}$;
$s$ - the thickness of the sheet material, mm;
$\sigma_{3 p}-$ cutting resistance (table value), $\mathrm{kg} / \mathrm{mm}^{2}$.
The actual cutting efforts in terms of production, taking into account factors are listed will be greater than estimated. Its value can be found using the coefficients that take into account the effect of each factor on the desired size. Total coefficient k , which is the sum of individual coefficients, that studies have shown, are in the range $1.0-1.3$. Then the real value of cutting force (punching) Rd, which is taken as the initial value when calculating the power to choose the equipment will be equal

$$
\begin{equation*}
P_{\partial}=k P=k u s \sigma_{3 p} . \tag{5}
\end{equation*}
$$

In the case where the manufacture of parts takes place in several transitions(operations), in this case it is necessary to calculate the cutting effort on each of them. The total (maximum) stamping effort is found as the sum of stamping effort at each of the transitions.

$$
\begin{equation*}
P_{\max }=P_{p 1}+P_{p 2}+\ldots+P_{p n} \tag{6}
\end{equation*}
$$

### 3.2. Determination of part push effort (removal).

a) In sheet punching pushing force depends mainly on the same factors as cutting efforts, that is why in practice to simplify calculation of push efforts of product through the matrix $\left(Q_{n p}\right)$ taken to determine the percentage of cutting efforts. Thus a pushing force of parts given by

$$
\begin{equation*}
Q_{n p}=k_{n p} P, \tag{7}
\end{equation*}
$$

where $k_{n p}$ - coefficient that taking into account the impact of various factors on push efforts similarly to the cutting effort.

The value of the coefficient can be selected from the table. 4
Table 4 - coefficient meaning

| Material | Material thickness $0,5-10 \mathrm{~mm}$ |
| :--- | :---: |
| Steel | $0,03-0,07$ |
| Copper and brass | $0,02-0,04$ |
| Aluminum | $0,03-0,05$ |
| duralumin | $0,03-0,07$ |

For thinner materials, the value $k_{n p}$ should be taken closer to the upper limit and for thicker ones - closer to the lower limit. For small gaps (from 5\% S and less) $Q_{n p}$ rise in 1.5-2.5 times; and for high gaps (over $20 \%$ S) $Q_{n p}$ approaches zero. In the presence of lubrication, the push force can be reduced by $20-40 \%$.

The actual pushing force will depend from number of parts $n$, which are simultaneously in the matrix hole.

If in a matrix the cylindrical belt has a height $h$, then in it can simultaneously be the number of parts equal:

$$
n=\frac{h}{S},
$$

where h is the height of the cylindrical belt of the matrix (selected constructively), mm ;
s - thickness of the part, mm;
In this case pushing effort will be equal:

$$
\begin{equation*}
Q_{n p}=k_{n p} P n, \tag{8}
\end{equation*}
$$

b) Removal effort of material from the punch $\left(Q_{3 и}\right)$ depends mainly from the same factors that push effort (with the exception of the shape of the matrix passage hole). In addition, when removing material from a punch, the size of the jumper has a significant effect (a) and the bend of the strip, which serves as a workpiece.

Removal efforts of material from the punch also determined by empirical relationships

$$
\begin{equation*}
Q_{3 u}=k_{3 H} P, \tag{9}
\end{equation*}
$$

where $k_{34}$ - coefficient that taking into account the impact of various factors on removal effort of material from punch.

The value of $k_{34}$, depending on the size of the jumper during the cutting of parts from the strip, and also for the punching holes in the detail are given in Table. 5

Table 5 - coefficient meaning $k_{34}$

| Material | Waste removing Part punching from tape | Part removing |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Hole punching in part |  |  |  |  | Punching of several holes |
|  |  | relation $\mathrm{a} / b^{*}$ |  |  |  |  |  |
|  |  | Up to 0,5 | $\begin{aligned} & 0,5 \div \\ & 1 \end{aligned}$ | $\begin{aligned} & 1 \div \\ & 1,5 \end{aligned}$ | $\begin{aligned} & 1,5 \div \\ & 2 \end{aligned}$ | $\begin{gathered} \hline \text { more } \\ 2 \end{gathered}$ |  |
| Steel | 0,03 | 0,04 | 0,045 | 0,05 | 0,06 | 0,07 | 0,07 |
| Brass, copper, zinc | 0,02 | 0,03 | 0,035 | 0,04 | 0,05 | 0,06 | 0,06 |
| aluminum, duralumin | 0,025 | 0,045 | 0,05 | 0,06 | 0,07 | 0,08 | 0,08 |

* where $a$ - gap value;
$b$-dimension of the hole that is punching.
The overall stamping effort required determine the work that is spent on cutting (punching) and equipment selection is determined by the formula

$$
\begin{equation*}
P_{3 a 2}=P_{\max }+Q_{n p}+Q_{34}, \tag{10}
\end{equation*}
$$

### 3.3. Calculation of spent work on stamping and power of the press

Work that is spent on cutting (punching), determined by the formula

$$
\begin{equation*}
A=\frac{\lambda \cdot P_{3 a 2} \cdot S}{1000}, \text { Дж } \tag{11}
\end{equation*}
$$

where $\lambda$ - coefficient expressing the ratio of average to maximum punching effort.
With a degree of accuracy sufficient for practice, the coefficient $\lambda$ can be chosen depending on the thickness of the material $S$ from the table. 6

Table 6 - The value of the coefficient $\lambda$ for determining the work involved in stamping

| Material thickness S, mm | Coefficien $\lambda$ |
| :--- | :---: |
| Less than 2 | $0,75-0,55$ |
| From 2 to 4 | $0,55-0,45$ |
| More than 4 | $0,45-0,30$ |

It should be noted that for soft and subtler materials it is necessary to take the value of the coefficient $\lambda$ closer to the upper boundary, and for the solids and thicker ones, the values are closer to the lower bound.

Knowing the amount of work A spent on the operation of cutting (punching), you can find, in known formulas, the required power of the press and electric motor.

Example. It is necessary to determine the effort and the work to be done to punch the part in the form of a rectangular plate (Fig. 7) with dimensions $b=$ $120 \mathrm{~mm}, \mathrm{c}=40 \mathrm{~mm}$ of steel of the Ct 3 with a thickness of 2.2 mm on a die, in which the angle of inclination of the cutting blades of the matrix is $\varphi=0^{\circ}$.


Figure 7 - Drawing details to determine the effort and the work involved in punching
For this brand of steel $\mathrm{Ct} 3, \sigma_{3 p}=350 \mathrm{mn} / \mathrm{m}^{2}\left(35 \mathrm{~kg} / \mathrm{mm}^{2}\right)$ tab 4 annex 1.
Then the cutting efforts with normal operating conditions

$$
P=u s \sigma_{3 p}=2(b+c) s \sigma_{3 p}=2(120+40) \times 2,2 \times 35=246400 N
$$

The real effort, for the choice of equipment, with the coefficient of correction $\mathrm{k}=1,3$

$$
P_{\partial}=k \times P=1,3 \cdot \times 24640=320320 H \approx 320 \kappa N
$$

Spent work on cutting at $\lambda=0,55$

$$
A=\frac{\lambda \cdot P_{3 P} \cdot S}{1000}=\frac{0,55 \cdot 24640 \cdot 2,2}{1000}=30 \mathrm{~kg} \cdot \mathrm{M}(300 \mathrm{~J})
$$

## 4. Stamp calculation and designing

### 4.1. Stamp pressure center determining

For stamp correct work it is necessary to combine pressure center with slider axis, that is provided by shank placement in stamp top plate in such a manner so its axis pass through the stamp pressure center. In other case band torque appears, as a result skewed of slide take place which leads to rapid wear of the stamp guide.

In practice there are two methods for stamp pressure center determining:

1) Analytical - method of deformation resistance moments;
2) graphical method.

In regular geometric shapes (contours) pressure centers coincide with their geometric center, so the pressure center of this contours is not required for additional calculations, and just use the known geometric equations. Example, for round contour pressure center will be on the center of the circle, for line - on the middle of it, for rectangular contours - at the intersection of the diagonals, etc.

In the complex contour coordinates of pressure center determine by static torque applied to harvested perimeter contour, which is divided into a number of elementary sections. After determining the center of all the plots resultant point of application are finding out.

Analytical method is as follows. Let we need to punch two holes with diameters $D_{1}, D_{2}$ (Fig. 8). At first we have to find the punching effort for each hole. The point of application of resultant force $P$ of forces $P_{1}$ and $P_{2}$ will be defining pressure center $O$. To establish a point of resultant force $P$ will make a balance equation, for example relative to the point $O_{l}$ relatively unknown quantity $x$ :


Figure 8 - Scheme for stamp pressure center determining with symmetrically placed holes

$$
\begin{gather*}
P_{2} l=\left(P_{1}+P_{2}\right) x, \\
x=\frac{P_{2} l}{P_{1}+P_{2}}, \tag{12}
\end{gather*}
$$

where
or based on punching effort values $P_{p 1}$ i $P_{p 2}$ :

$$
\begin{equation*}
x=\frac{k u_{2} s \tau_{o} l}{k\left(u_{1}+u_{2}\right) s \tau_{o}}=\frac{u_{2} l}{u_{1}+u_{2}}=\frac{\pi D_{2} l}{\pi\left(D_{1}+D_{2}\right)}=\frac{\pi D_{2} l}{D_{1}+D_{2}} \tag{13}
\end{equation*}
$$

That is, for practical determination of stamp pressure center position is enough to know the linear dimensions of punches and their relative positions.

In case of simultaneous cutting of dissimilar holes randomly located relative to both axes (Fig. 9) coordinates of stamp pressure center are determined by the equation:

$$
\begin{align*}
& x=\frac{P_{1} x_{1}+P_{2} x_{2}+P_{3} x_{3}+\ldots+P_{n} x_{n}}{P_{1}+P_{2}+P_{3}+\ldots+P_{n}} ;  \tag{14}\\
& y=\frac{P_{1} y_{1}+P_{2} y_{2}+P_{3} y_{3}+\ldots+P_{n} y_{n}}{P_{1}+P_{2}+P_{3}+\ldots+P_{n}} . \tag{15}
\end{align*}
$$

Expressing these formulas through the length (perimeter) of harvested contours we will get:

$$
\begin{equation*}
x=\frac{u_{1} x_{1}+u_{2} x_{2}+u_{3} x_{3}+\ldots+u_{n} x_{n}}{u_{1}+u_{2}+u_{3}+\ldots+u_{n}} ; \tag{16}
\end{equation*}
$$

$$
\begin{equation*}
y=\frac{u_{1} y_{1}+u_{2} y_{2}+u_{3} y_{3}+\ldots+u_{n} y_{n}}{u_{1}+u_{2}+u_{3}+\ldots+u_{n}} . \tag{17}
\end{equation*}
$$



Figure 9 - Scheme for stamp pressure center determining with holes randomly placed relative to both coordinate axes

Example: It is necessary to determine the pressure center when punching the contour (Fig. 1).

1. At an arbitrary distance from the contour, we construct a rectangular coordinate system XY (Fig. 11).
2. 2. The perimeter (contour) of the details is divided into elementary areas $1_{1}$, $1_{2}, \ldots, 1_{8}$ and we determine their centers of gravity.
In rectilinear areas, centers of gravity are placed in the middle of the segments. Centers for the arcs weight of circles with radius $\mathrm{R}_{1}=30 \mathrm{~mm}$, and $\mathrm{R}_{2}=40 \mathrm{~mm}$ located on axes of symmetry of arcs AB and CD , and their distance from the centers of radius C 1 and C 2 according to the statics of solids bodies, is determined by the formula:

$$
C=\frac{2 \sqrt{2} \cdot R}{\pi} \approx 0,9 \cdot R
$$

Where $\mathrm{R}_{1}=30 \mathrm{~mm}, \mathrm{C}_{1}=27 \mathrm{~mm}$; при $\mathrm{R}_{1}=40 \mathrm{~mm}, \mathrm{C}_{1}=36 \mathrm{~mm}$.
To determine the center of gravity of the considered arcs relatively to the axes X and Y , it is necessary to find the auxiliary sizes $\mathrm{a} 1, \mathrm{~b} 1, \mathrm{a} 2, \mathrm{~b} 2$ as cathetus of rectangular triangles (Fig. 11).
3. In accordance with the drawing of the part, we determine the coordinates of the pressure centers of all elementary areas relative to the coordinate axes. Before, we take the distances of the outer sides (or angles) of the contour to the coordinate axes equal to 20 mm .
4. Determine the coordinates of the pressure center of the entire contour according to the formulas:
relative to the Y axis:

$$
\begin{equation*}
X_{0}=\frac{l_{1} \cdot X_{1}+l_{2} \cdot X_{2}+\ldots+l_{n} \cdot X_{n}}{l_{1}+l_{2}+\ldots+l_{n}} ; \tag{18}
\end{equation*}
$$

relative to the X axis:

$$
Y_{0}=\frac{l_{1} \cdot Y_{1}+l_{2} \cdot Y_{2}+\ldots+l_{n} \cdot Y_{n}}{l_{1}+l_{2}+\ldots+l_{n}},
$$

where, $\mathrm{X}_{1}, \mathrm{X}_{2}, \ldots \mathrm{X}_{\mathrm{n}}$ - abscissa of centers of weight of elements of a contour;
$\mathrm{Y}_{1}, \mathrm{Y}_{2}, \ldots \mathrm{Y}_{\mathrm{n}}$ - ordinates of centers of weight of contour elements.
The length of the elementary areas of the contour is based on the dimensions of the drawing of the part:

$$
\begin{aligned}
l_{1}=\sqrt{10^{2}+40^{2}} \approx 41 \mathrm{~mm} ; \quad l_{2}=180-30=150 \mathrm{~mm} ; \quad l_{3}=\frac{2 \cdot \pi \cdot 30}{4}=47 \mathrm{~mm} \\
l_{4}=80-30-20=30 \mathrm{~mm} \quad l_{5}=\sqrt{20^{2}+40^{2}}=45 \mathrm{~mm} \\
l_{6}=180-40-40-40-10=50 \mathrm{~mm} ; \quad l_{7}=\frac{2 \cdot \pi \cdot 40}{4}=63 \mathrm{~mm} l_{8}=40 \mathrm{~mm}
\end{aligned}
$$

After substitution of the found values in the formulas 18,19 we obtain:

$$
\begin{aligned}
& X_{0}=\frac{41 \cdot 195+150 \cdot 125+47 \cdot 31+30 \cdot 20+45 \cdot 40+50 \cdot 85+63 \cdot 124,5+40 \cdot 170}{466}=\frac{49496}{466}=106 \mathrm{~mm} \\
& Y_{0}=\frac{41 \cdot 40+150 \cdot 20+47 \cdot 31+30 \cdot 65+45 \cdot 90+50 \cdot 100+63 \cdot 74,5+40 \cdot 60}{466}=\frac{21190}{466}=52 \mathrm{~mm} ;
\end{aligned}
$$



Fig 1. Drawing of the detail


Fig 2. circuit for determining the center of pressure of a difficult cutting contour

### 4.2. Determination of the closed height of the stamp.

The stamp is projected in its lower working position. In this position, in the best way bound the interaction of working (punch and matrix), clamping guides and guideway or removing knots of detail and stamp.

The height of the stamp in its lower working position is called the closed height of the stamp, which must necessarily be matched to the closed height of the press.

The closed height of the press is called the distance from the slab to the press slide block in its lower position at the maximum travel and shortest length of the connecting rod. Closed height of the press indicates the maximum height of the stamp, which can be installed, to work on this press.

Stamp that has a closed high bigger than the closed height of the press, can`t be installed on this press. And if it is installed, in the event that the slider of the press will be in the extreme upper position, then at the press start its breakage may occur.

For presses with adjustable stroke closed heights are accepted in case of work at the maximum travel of the press. When decreasing the distance of travel closed height increases proportionally to the ratio.

$$
\left(\frac{h_{\max }-h}{2}\right)
$$

The technical characteristics of the press includes three groups of indicators: the parameters that determine the technological capabilities of the press, the size of the places for the establishment of stamps, which determine the dimensions of the stamp and the size of the elements of its connection to the press; constructive characteristics.

In presses with constant motion of a slider and a fixed table the limits of change of the closed height of a stamp are determined only by the size of regulation of a length of a connecting rod: the closed height of a stamp
the biggest

$$
\begin{align*}
& H_{\max }=H-H_{n \pi}  \tag{20}\\
& H_{\min }=H-H_{n \lambda}-\Delta_{u} \tag{21}
\end{align*}
$$

the smallest
On presses with a movable table and a variable slider movemens, if the press table is lowered to the lower position, the connecting rod is adjusted to the smallest length and the stroke of the press is the smallest, a stamp with the largest closed height can be installed.

$$
\begin{equation*}
H_{u}^{\max }=H-H_{n z}+\frac{h_{\max }-h_{\min }}{2} \tag{22}
\end{equation*}
$$

If the press table is raised to the top position, the connecting rod is adjusted to the greatest length and the stroke of the press is the largest, then the press can be fitted with a stamp with the smallest closed height

$$
\begin{equation*}
H_{u}^{\min }=H-H_{n z}-\Delta_{u}-\Delta_{c}, \tag{23}
\end{equation*}
$$

where N - nominal closed height of the press, mm;
$H_{n, ~}$ - thickness of the plat under the stamp, mm;
$\Delta_{u}$ - the value of adjusting the length of the connecting rod (for dual-action presses - connecting rods), mm;
$\Delta_{c}$ - value of the table adjustment, mm;
$h_{\text {min }}, h_{\text {max }}$ - respectively, the smallest and largest movement (internal and external) of the slider, mm .

Within the limits between $H_{u *}^{\text {max }}$ and $H_{u}^{\text {min }}$ are all intermediate values of the closed height of the stamps installed on this press.

At some given slider movement $h_{0}$ a stamp can be installed on the press with closed height the value of which is within the range from

$$
\begin{equation*}
H_{0}^{\min }=H-H_{n v}-\Delta_{u}-\Delta_{c}+\frac{h_{\max }-h_{0}}{2} \tag{24}
\end{equation*}
$$

to

$$
\begin{equation*}
H_{0}^{\max }=H-H_{n z}+\frac{h_{\max }-h_{0}}{2} . \tag{25}
\end{equation*}
$$

If $h_{0}=h_{\text {max }}$ then

$$
\begin{gather*}
H_{u u}^{\min }=H_{0}^{\min }=H-H_{n z}-\Delta_{u u}-\Delta_{c},  \tag{26}\\
H_{0}^{\max }=H-H_{n t} \tag{27}
\end{gather*}
$$

The last value of the closed height of the stamp corresponds to the nominal closed height of the press.

Example: The press of the 25 Tf model of the KD 1424 has the following technological parameters: $H=450 \mathrm{~mm}, \Delta_{u}=55 \mathrm{~mm} ; \Delta_{c}=270 \mathrm{~mm} ; h_{\text {мax }}=65 \mathrm{~mm} ; h_{\text {мin }}=$ $5 \mathrm{~mm} ; H_{n \pi}=50 \mathrm{~mm}$. t is necessary to determine the possible closed height of the die mounted on this press while movement of the press slider $h_{o}=25 \mathrm{~mm}$.

The highest closed stamp height is:

$$
\begin{array}{r}
H_{0}^{\max }=H-H_{n z}+\frac{h_{\max }-h_{0}}{2}=450-50+\frac{65-25}{2}=420 \mathrm{~mm} \\
H_{0}^{\min }=H-H_{n z}-\Delta_{w}-\Delta_{c}+\frac{h_{\max }-h_{0}}{2}=450-50-55-270+\frac{65-25}{2}=95 \mathrm{~mm}
\end{array}
$$

### 4.3. Selection of gaps and determination of executive (working) dimensions of

 the punch and matrix.In determining the operating (working) sizes of punches and matrices for disconnection operations of sheet stamping it is necessary to proceed from the dimensions of the stamped part, its accuracy and the parameter of stamp durability. To obtain a stamped part with a given degree of accuracy, it is necessary to provide the correct choice (appointment) of gaps and tolerances on the working dimensions of the punch and matrix. In this case, the tolerances on the working dimensions of punches and matrices should be set in such limits that they provide the best gaps.

Setting the gap direction. The size of the cut out part, in the outer contour, depends on the size of the matrix. The size of the opening that breaks - from the size of the punch. Based on this we can set the gap direction, depending on which parts of the component need to hold - external or internal (covering or covered). In the case of
cutting out the external contour, the nominal dimensions of the part $\mathrm{D}_{\mathrm{H}}$ are provided by the matrix, and the gap $z$ is provided by reducing the size of the punch $D_{n}$, it means $D_{n}=D_{n_{-}}-z$. The nominal size of the matrix is assumed to be equal to the smallest dimensional dimension of the part. When punching the holes, its dimensions $d_{n}$ are provided to the punch $d_{n}$, and the gap $z$ is provided by increasing the size of the matrix $d_{\mu}$, it means $d_{N}=d_{H}+z$.

Appointment (selection) of the gaps size. The value of the minimum and maximum initial bilateral gaps for various materials when working on presses with a number of moves $120-140$ per minute are given in Table. 6 or in the table. 7. The smallest initial gaps are nominal. The largest initial gaps take into account their increases through the tolerances for the manufacture of a punch and matrix.

Table 7 - Minimal and maximal initial bilateral gaps between the matrix and the punch in stamps for cutting and punching

| Material thickness $S$, мм | Material |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Soft steel (mark 08, 10, 15, Ст. 1, Ст. 2), copper, soft brass, aluminum |  | Steel of medium hardness (mark 20, 25, 30, 35, Ст. 3 , Ст.4), hard brass |  | Hard steel (mark 40, 45, 50, Ст. 5, Ст. 6 i вище), hard bronze |  |
|  | $z_{\text {min }}$ | $z_{\text {max }}$ | $z_{\text {min }}$ | $z_{\text {max }}$ | nin | $z_{\text {max }}$ |
| 0,5 | 0,020 | 0,040 | 0,025 | 0,050 | 0,030 | 0,055 |
| 0,6 | 0,025 | 0,050 | 0,030 | 0,060 | 0,040 | 0,070 |
| 0,8 | 0,030 | 0,065 | 0,040 | 0,080 | 0,050 | 0,090 |
| 1,0 | 0,040 | 0,080 | 0,050 | 0,100 | 0,060 | 0,110 |
| 1,2 | 0,060 | 0,120 | 0,070 | 0,130 | 0,080 | 0,160 |
| 1,5 | 0,075 | 0,140 | 0,090 | 0,165 | 0,100 | 0,195 |
| 1,8 | 0,090 | 0,160 | 0,110 | 0,200 | 0,130 | 0,230 |
| 2,0 | 0,100 | 0,180 | 0,120 | 0,220 | 0,140 | 0,260 |
| 2,5 | 0,125 | 0,225 | 0,150 | 0,275 | 0,175 | 0,325 |
| 3,0 | 0,150 | 0,270 | 0,180 | 0,330 | 0,210 | 0,390 |
| 3,5 | 0,210 | 0,350 | 0,245 | 0,420 | 0,280 | 0,490 |
| 4,0 | 0,240 | 0,400 | 0,280 | 0,480 | 0,320 | 0,560 |
| 4,5 | 0,270 | 0,450 | 0,315 | 0,540 | 0,360 | 0,630 |
| 5,0 | 0,300 | 0,500 | 0,350 | 0,600 | 0,400 | 0,700 |
| 6,0 | 0,400 | 0,660 | 0,500 | 0,800 | 0,500 | 0,900 |
| 7,0 | 0,500 | 0,770 | 0,600 | 0,900 | 0,600 | 1,100 |
| 8,0 | 0,600 | 0,880 | 0,700 | 1,100 | 0,700 | 1,200 |
| 9,0 | 0,700 | 1,000 | 0,800 | 1,300 | 0,900 | 1,400 |
| 10 | 0,800 | 1,200 | 0,900 | 1,400 | 1,000 | 1,600 |
| 12 | 1,000 | 1,500 | 1,100 | 1,700 | 1,200 | 2,000 |

Tolerances on the executive (working) dimensions of the punch and matrix.
a) when cutting and punching a round profile.

In fig. 12 shows the schemes of stamping and placement of tolerance fields on the executive (working) dimensions of the punch and matrix when cutting a round hole.

Taking into account that the run-out of the matrix leads to an increase in its size and the run-out of the punch - to reduce, then their nominal sizes are given respectively: the minimum - for the matrix and the maximum - for the punch.

The dimensions of the punch and the matrix are determined by the following dependencies (see diagram):
when cutting out the outer contour:

$$
\begin{gather*}
D_{M}=\left(D_{H}-\Delta^{\prime}\right)^{+\delta_{M}}  \tag{28}\\
D_{\Pi}=\left(D_{H}-\Delta^{\prime}-z_{\min }\right)_{-\delta_{\Pi}} \tag{29}
\end{gather*}
$$

when punching the hole:

$$
\begin{gather*}
d_{M}=\left(d_{H}-\Delta^{\prime}+z_{\min }\right)^{+\delta_{M}}  \tag{30}\\
d_{\Pi I}=\left(d_{H}-\Delta^{\prime}\right)_{-\delta_{\Pi}} \tag{31}
\end{gather*}
$$

where $D_{n}, D_{\mu}, d_{n,} d_{\mu}$ - respectively the diameters of punches and matrices when cutting and punching;
$D_{H}, d_{H}$ - nominal diameters of the parts;
$z_{\text {мін }}$ - minimum (guaranteed) gap between the matrix and the punch;
$\Delta$ - the tolerance field of the stamped part (on the outer contour or hole), which is given by the drawing;
$\Delta^{\prime}$ - allowance for wear of the tool;
$\delta_{n}$ and $\delta_{m}$ - accordingly tolerance for the manufacture of the punch and the matrix:

In determining the operating (working) dimensions of the stamp should also take into account the allowance for the wear of the punch and the matrix $\Delta^{\prime}$, which is determined depending on the required accuracy of the stamping of the part: if the tolerance on the size of the part $\Delta \leq 100$ microns, then the allowance for wear is taken $\Delta^{\prime}=\Delta$, in the case of $\Delta>100$ microns, then $\Delta^{\prime}=0,8 \Delta$ (the scheme of placement of tolerance fields in Fig. 12 is given at $\Delta^{\prime}=\Delta$ ).

Total tolerances of tolerances for the manufacture of the matrix and punch should not exceed the clearance on the gap

$$
\delta_{n}+\delta_{M} \leq\left(z_{\text {мах }}-z_{\text {міп }}\right) .
$$

The tolerances for the operating (working) dimensions of the matrix and the punch $\delta_{n}$ and $\delta_{M}$ are taken on the 7th grade of accuracy when cutting (punching) of parts with the thickness $S \leq 4 \mathrm{~mm}$, at $S>4 \mathrm{~mm}$, it is possible to take on the 8th grade.


Figure 3 - scheme of stamping and placement of tolerance fields on the executive (working) dimensions of the punch and matrix during the cutting and making a round contour
a) when cutting out the external contour
b) when breaking the hole

Example: It is necessary to calculate the executive (working) dimensions of the punch and the matrix and assign tolerances to these dimensions for the washer shown in Fig. 13. Material - steel 35.

Tolerances on the dimensions of the parts are:
$\Delta 60=300$ microns $=0.3 \mathrm{~mm} ; \Delta 30=210$ microns $=0.21 \mathrm{~mm}$.


Figure 13 - Drawing of the detail for determination and calculation of executive (working) dimensions of the punch and matrix

Since the tolerances for the sizes $\Delta>100$ microns, the tolerances for wear on the corresponding working matrixes and the punch on these sizes are equal

$$
\begin{aligned}
& \Delta_{60}^{\prime}=0,8 \times \Delta_{60}=0,24 \mathrm{~mm} ; \\
& \Delta_{30}^{\prime}=0,8 \times \Delta_{60}=0,17 \mathrm{~mm} .
\end{aligned}
$$

According to Table 7 for the thickness of the material $\mathrm{S}=2.2 \mathrm{~mm}$ (we take the nearest smaller value -2.0 mm ) we need to take $\mathrm{Z}_{\text {min }}=0,12 \mathrm{~mm}$.

In addition, according to tolerance tables for sizes $\varnothing 60 \mathrm{~mm}$ and $\emptyset 30 \mathrm{~mm}$ for the 7 th qualification, we find the corresponding values $\delta_{\mathrm{H}}$ and $\delta_{\mathrm{M}}$ :

$$
\begin{array}{ll}
\delta_{\mathrm{M} 60}=0,030 \mathrm{~mm} ; & \delta_{\text {H } 60}=0,030 \mathrm{~mm} ; \\
\delta_{\mathrm{M} 30}=0,021 \mathrm{~mm} ; & \delta_{\text {H } 30}=0,021 \mathrm{~mm} .
\end{array}
$$

We find the values of the dimensions of the matrix and the punch for the external contour, taking into account the allowance for wear according to the formulas 28, 29:

$$
\begin{gathered}
D_{M}=\left(D_{H}-\Delta^{\prime}\right)^{+\delta_{M}}=(60-0,24)^{+0,03}=59,76^{+0,03} \mathrm{~mm} ; \\
D_{I I}=\left(D_{H}-\Delta^{\prime}-z_{\min }\right)_{-\delta_{n}}=(60-0,24-0,12)_{-0,03}=59,64_{-0,03} \mathrm{~mm} .
\end{gathered}
$$

For an aperture according to formulas 30,31 we find

$$
\begin{gathered}
d_{M}=\left(d_{H}-\Delta^{\prime}+z_{\min }\right)^{+\delta_{M}}=(30+0,17+0,12)-_{0,021}=30,17_{-0,021} \mathrm{~mm} \\
d_{\Pi}=\left(d_{H}-\Delta^{\prime}\right)_{-\delta_{\Pi}}=(30+0,17)^{+0,021}=30,29^{+0,21} \mathrm{~mm} .
\end{gathered}
$$

### 4.4. Definition of stamp stability

On the stamping resistance have significantly influence stamping material, the thickness of the stamped material, the configuration and the dimensions of the part, the design features of the stamp, the material and the heat treatment of the parts of the stamp, the operating conditions.

During the elaboration of this sub-clause it is necessary to carry out a checking calculation of the punch, matrix or other critical elements of the equipment for durability in the case of significant stamping efforts, openings of small diameters and with considerable thickness (hardness) of the material. The material of the tool can be steel with HRC $56 \div 60, \sigma_{c m}=1000 \div 1600 \mathrm{MPa}$.

To penetrate the condition of durability of the punch has the form:

$$
\begin{equation*}
\sigma_{c m}=P_{p 1} / F \leq[\sigma]_{c m}(\text { мПа }) \tag{32}
\end{equation*}
$$

where F - is the area of the smallest cross-section of the punch, $\mathrm{m}^{2}$;
$\sigma_{c m}$ - compression stress in the punch, MPa ;
$[\sigma]_{c m}-$ allowable compressive stresses for hardened steel, mPa .
Pressure on gaskets of a stamp from a punch (matrix):

$$
\begin{equation*}
p=\frac{P_{\max }}{F_{b}}=\frac{k u s \tau_{o}}{F_{b}} \leq[\sigma]_{3 M}(\mathrm{mPa}) \tag{33}
\end{equation*}
$$

where $F_{b}$ - sectional area of the upper part of the punch (matrix), $\mathrm{m}^{2}$;
$[\sigma]_{3 M}$ - permissible tensile strength of the stamp plates material, mPa .
Check for matrix strength:
a) a circular matrix with a diameter $\mathrm{d}=2 \mathrm{r}$, which rests on a plate with an internal diameter $\mathrm{d}_{0}=2$ ro. Bending tension

$$
\begin{equation*}
\sigma_{32}=\frac{2,5 P_{\max }}{H_{m}^{2}}\left(1-\frac{2 r}{3 r_{o}}\right) \leq[\sigma]_{32}(\mathrm{mPa}) \tag{34}
\end{equation*}
$$

matrix thickness

$$
\begin{equation*}
H_{m}=\sqrt{\frac{2,5 P_{\max }}{[\sigma]_{3 M}}\left(1-\frac{2 r}{3 r_{o}}\right)}(\mathrm{mm}) \tag{35}
\end{equation*}
$$

b) a rectangular matrix based on a plate with a square aperture with side $a$

$$
\begin{gather*}
\sigma_{32}=\frac{2,5 P_{\max }}{H_{m}^{2}} \leq[\sigma]_{32}(\mathrm{mPa})  \tag{36}\\
H_{m}=\sqrt{\frac{2,5 P_{\max }}{[\sigma]_{3 M}}}(\mathrm{~mm}) \tag{37}
\end{gather*}
$$

c) a rectangular matrix based on a plate with a rectangular aperture with sides $a \times b$

$$
\begin{equation*}
\sigma_{32}=\frac{3 P_{\max }}{H_{m}^{2}}\left(\frac{b / a}{1+b^{2} / a^{2}}\right) \leq[\sigma]_{32}(\mathrm{mPa}) \tag{38}
\end{equation*}
$$

$$
\begin{equation*}
H_{m}=\sqrt{\frac{3 P_{\max }}{[\sigma]_{3 n}}\left(\frac{b / a}{1+b^{2} / a^{2}}\right)}(\mathrm{mm}), \tag{399}
\end{equation*}
$$

where $[\sigma]_{3 \mu}$ - permissible bending stress, mPa .

### 4.5. Calculation of structural elements of the stamp

For pressing the material of the workpiece to be stamped, for removal of a product from the punch, for pushing the product from the matrix using different clamps, strippers, called buffers. The buffers can be rubber, polyurethane, metal elastic elements, as well as pneumatic and hydraulic devices. Last ones are expensive, and therefore in the press of a simple action it is recommended to use the first types of buffer devices.

An important condition for calculation of the buffer spring is that the removal force of the product should be less than effort that breaks the buffer $Q_{3}<P_{\bar{\sigma}}$.

Screw cylindrical springs for buffers can be made from a wire of round, square, rectangular cross-section. The material for the springs is spring steel grades $65 \Gamma$, $60 \mathrm{C} 2,60 \mathrm{C} 2 \mathrm{~A}$. Springs harden in oil with a release, after which the hardness is HRC $38-45$.

The developing buffer force:

- for springs with a round cross-section

$$
\begin{equation*}
P_{\bar{\sigma}}=\frac{\pi d^{3}[\tau]_{k p}}{8 D_{c p}}(\mathrm{H}), \tag{40}
\end{equation*}
$$

where $d$ - diameter of the cross-section of the spring, mm ;
$D_{c p}$ - average diameter of the spring, mm;
$[\tau]_{k p}-$ allowable torsion stress ( 500 MPa for steel $65 \Gamma ; 550-650 \mathrm{MPa}$ for steels $60 \mathrm{C} 2,60 \mathrm{C} 2 \mathrm{~A}$ ).

- for springs with rectangular section of turns

$$
\begin{equation*}
P_{\bar{\sigma}}=0,416 \frac{c^{3}[\tau]_{k p}}{D_{c p}}(\mathrm{H}), \tag{41}
\end{equation*}
$$

where c - width of a turn, mm .

- for a combined spring

$$
\begin{equation*}
\sum P=P_{1}\left(1+\frac{d_{2}^{2}}{d_{1}^{2}}+\frac{d_{3}^{2}}{d_{1}^{2}}+\ldots+\frac{d_{n}^{2}}{d_{1}^{2}}\right)(\mathrm{H}) . \tag{42}
\end{equation*}
$$

Plate spring allow to create significant efforts at small dimensions. The total compression force is equal to the sum of the forces of individual springs. Material of the springs is steel 60C2A, heat treatment - tempering with the release to HPC 46-50. Effort which create plate spring

$$
\begin{equation*}
P_{\bar{\sigma}}=\frac{200 \cdot s^{2}(1+\operatorname{tg} \alpha)^{2}}{\sqrt[8]{n}(2,4-2 d / D)}(\mathrm{H}), \tag{43}
\end{equation*}
$$

where $s$ - the thickness of the spring washer;
$\alpha$ - angle of inclination of the surface of the spring washer, $\alpha=4 \div 7^{\circ}$;
$n$ - number of springs;
$d$ - inner diameter of the spring;
$D$ - outer diameter of the spring.
Rubber and polyurethane are used for strippers, pushers and pressers in die stamping machines. The shape and dimensions of rubber and polyurethane buffers are chosen depending on the required effort. Material of springs: sheet technical oil-and-gas-resistant rubber, hardness of rubber $50-70$ by Shore; Polyurethane brand SKU-7L, hardness of polyurethane 76-86 by Shore.

Effort which create rubber (polyurethane) buffer

$$
\begin{equation*}
P_{\bar{\sigma}}=p S_{o n}(\mathrm{H}), \tag{44}
\end{equation*}
$$

where $p$ - pressure when compressing the buffer, MPa;
$S_{o n}$ - bearing buffer area, $\mathrm{cm}^{2}$.
For a rubber buffer, maximum compression is recommended to take no more than $45 \%$ (preferably $25 \div 30 \%$ ).

After selecting a buffer device, be sure to check the condition $Q_{з и}<P_{\sigma}$ and give the scheme of placement of the device with indication of its constructive parameters.
4.6. Description of the structure and the principle of the stamp

According to the previous analysis and calculations it is necessary to develop a technological scheme of stamping the product with the following graphic image and describe the implementation of the technological process.

## 5. Choice of equipment and its technical characteristics

According to preliminary calculations for maximum stamping effort $P_{\max }$, dimensional details and other structural and power parameters in finally accepted model of press equipment. Choose the main design and technological characteristics of the equipment, the results need to draw in the form of a table.

## Conclusion

In conclusion, to the course work should indicate the results of developments, recommendations for improving the technological process of manufacturing this product by sheet stamping.

## APPENDICES

Appendix 1
The value of the resistance to cutting (punching) $\tau_{0}$ for the most common metals [4]

| Material | Soft metal <br> (annealed) | Solid metal <br> (clutched) |
| :--- | :---: | :---: |
|  | $\tau_{0}$, МПа | $\tau_{0}$, MПа |
| Steel 08 | $250-280$ | $320-350$ |
| Steel 10кп; 15кп; Ст1 | $280-300$ | $350-380$ |
| Steel 20; Ст2 | $300-320$ | $380-420$ |
| Steel 25; Ст3 | $320-350$ | $420-450$ |
| Steel 30; Ст4 | $350-380$ | $450-500$ |
| Steel 35; Ст5 | $400-450$ | $500-550$ |
| Steel 40-45; Ст6 | $450-500$ | $550-580$ |
| Stainless steel | 520 | 560 |
| Stainless steels 12X18Н9; 12X13 | $360-380$ | - |
| ball-bearing steel ШX15 | $460-520$ | - |
| Copper M1; М2; М3 | $180-220$ | $250-280$ |
| Brass Л63; Л68 | $220-280$ | $350-400$ |
| Aluminum АД; АД1 | $70-90$ | $110-150$ |
| Duralumin Д1; Д16 | $140-180$ | $260-380$ |

Annex 2

Basic characteristics of crankshaft specifications [9]

| Technological characteristics |  |
| :--- | :--- |
| Stamp effort, thousands | $P$ |
| Inner and outer sliders stroke: - minimum, mm <br> - maximum, mm | $h_{\min }$ <br> $h_{\max } h$ |
| Frequency of slider movement, rpm | $n$ |
| The nominal closed height, mm | $H$ |
| Nominal closed height of external slider of double action press, mm | $H_{H a p}$ |
| Thickness of under stamp plate, mm | $H_{n \lambda}$ |
| Connecting rod length adjust, mm | $A_{u}$ |
| Adjusting the table, mm | $U_{c}$ |
| Overhang, mm | $R, R^{\prime}$ |
| The distance between the guides, mm | $L_{H}$ |
| The distance between racks, mm | $L_{C}$ |
| The greatest stroke of toper ejector, mm | $h_{6 . B}$ |
| The greatest stroke of bottom ejector, mm | $h_{H . B}$ |
| efforts to lower ejector, thousands | $P_{H . B}$ |
| clamp efforts, thousands | $Q$ |


| The sizes of windows in racks: - width, мм <br> - height, мм <br> - windows height of above the table, mm | $\begin{aligned} & A_{o} \\ & H_{o} \\ & l_{o} \end{aligned}$ |
| :---: | :---: |
| Technological work, kgp•m | $A$ |
| The largest weight of stamp that is hung to slider, kg | $m_{u}$ |
| Sizes of stamp installation places |  |
| Table dimensions: - length, mm - width, mm | $\begin{array}{\|l\|l} \hline A_{c} \\ B_{c} \\ \hline \end{array}$ |
| The dimensions of the hole in the table: - diameter, mm <br> - length, mm <br> - width, mm | $\begin{aligned} & D_{c} \\ & a_{c} \\ & b_{c} \end{aligned}$ |
| The dimensions of the hole in the under stamp plate, mm | $\begin{aligned} & D_{n ı} \\ & d_{n l} \\ & h_{n n} \end{aligned}$ |
| The dimensions of slider (double action presses - internal slider): <br> - length, mm <br> - width, mm | $\begin{aligned} & A_{n 3} \\ & B_{n 3} \end{aligned}$ |
| The dimensions of the central hole in slider (double action presses - in the inner slider) for fixing the top of the stamp, mm | $\begin{aligned} & d_{n 3} \\ & l_{n 3} \\ & \hline \end{aligned}$ |
| Design characteristics |  |
| Overall dimensions of machine:- length, mm  <br>  - width, mm <br>  - height above the floor, mm <br>  - level of the table, mm | $\begin{aligned} & \hline A_{\Gamma} \\ & B_{\Gamma} \\ & H_{\Gamma} \\ & H_{c} \\ & \hline \end{aligned}$ |
| Main drive motor power, kW | $N_{\text {гл }}$ |

Technological parameters of the simple-action stamps

| One-rack one-crank with movable table |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Model | $P$ | $h_{\text {min }}$ | $h_{\max }$ | $n$ | $H$ | $H_{n n}$ | $H_{p}$ | $\Delta_{u}$ | $\Delta_{c}$ | $R$ | $R^{\prime}$ |  |  |  |  |  |  |  |
| КД1424 | 25 | 5 | 65 | 120 | 450 | 50 | 210 | 55 | 270 | 190 | 180 |  |  |  |  |  |  |  |
| КД1426 | 40 | 10 | 80 | 100 | 500 | 65 | 250 | 65 | 300 | 220 | 210 |  |  |  |  |  |  |  |
| КД1428 | 63 | 10 | 100 | 90 | 530 | 80 | 280 | 80 | 290 | 260 | 250 |  |  |  |  |  |  |  |
| К1430 | 100 | 25 | 130 | 40 | 560 | 100 | 320 | 100 | 280 | 320 | 310 |  |  |  |  |  |  |  |
| КA1432A | 160 | 20 | 120 | 70 | 600 | 120 | 360 | 120 | 260 | 360 | 350 |  |  |  |  |  |  |  |

One-rack with unmovable table

| Model | $P$ | $h_{\text {min }}$ | $h_{\max }$ | $n$ | $H$ | $H_{n \lambda}$ | $\Delta_{u}$ | $R$ | $L_{c}$ |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| К2114 | 2,5 | 4 | 30 | 200 | 150 | 25 | 25 | 95 | 120 |
| КД2114 | 2,5 | 4 | 36 | 200 | 180 | 36 | 32 | 100 | 130 |
| К2116Б | 4,0 | 45 | - | 320 | 160 | 32 | 32 | 100 | 130 |
| КД2118 | 6,3 | 5 | 50 | 150 | 200 | 45 | 40 | 150 | 140 |
| К2118Б | 6,3 | 5 | 45 | 150 | 170 | 32 | 32 | 110 | 120 |


| КД2120 | 10 | 5 | 50 | 120 | 200 | 32 | 40 | 130 | 170 |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| КД2122 | 16 | 5 | 55 | 120 | 220 | 40 | 45 | 160 | 200 |
| КД2I24 | 25 | 5 | 65 | 120 | 450 | 50 | 55 | 190 | 240 |
| КД2126 | 40 | 10 | 80 | 100 | 280 | 65 | 65 | 220 | 280 |
| КД2128 | 63 | 10 | 100 | 90 | 340 | 80 | 80 | 260 | 340 |
| К2130 | 100 | 25 | 130 | 80 | 400 | 100 | 100 | 320 | 400 |
| КЕ2130 | 100 | 10 | 130 | 100 | 400 | 100 | 100 | 340 | 400 |
| К2132 | 160 | 25 | 160 | 70 | 480 | 120 | 120 | 360 | 480 |
| К2132A | 160 | 25 | 160 | 50 | 480 | 120 | 120 | 360 | 480 |
| К2232 | 160 | 160 | - | 37 | 480 | 120 | 120 | 360 | 480 |
| К2234 | 250 | 200 | - | 35 | 500 | 140 | 140 | 400 | 500 |
| К0134 | 250 | 200 | - | 32 | 560 | 140 | 140 | 400 | - |

Dimensions of places for stamp installing in single action crank press

| Model | $A_{c}$ | $B_{c}$ | $D_{c}$ | $a_{c}$ | $b_{c}$ | $D_{n \lambda}$ | $d_{n \lambda}$ | $h_{n \lambda}$ | $A_{n 3}$ | $B_{n 3}$ | $d_{n 3}$ | $l_{n 3}$ |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| КД1424 | 500 | 340 | 210 | 250 | 170 | 130 | 100 | 20 | 280 | 225 | 40 | 60 |
| КД1426 | 600 | 400 | 250 | 300 | 200 | 130 | 100 | 20 | 350 | 285 | 50 | 70 |
| КД1428 | 710 | 480 | 300 | 360 | 240 | 170 | 140 | 30 | 370 | 310 | 50 | 75 |
| К1430 | 850 | 560 | 360 | 420 | 280 | 240 | 200 | 30 | 330 | 295 | 60 | 85 |
| КА1432А | 1000 | 670 | 420 | 480 | 320 | 240 | 200 | 40 | 460 | 560 | 75 | 85 |
| К2114 | 260 | 170 | 90 | - | - | 70 | 50 | 15 | 110 | 80 | 25 | 50 |
| КД2114 | 280 | 180 | 90 | - | - | 70 | 50 | 15 | 120 | 95 | 25 | 55 |
| К2116Б | 280 | 180 | 120 | 140 | 90 | 70 | 50 | 15 | 95 | 95 | 25 | 45 |
| КД2118 | 360 | 280 | 120 | 150 | 100 | 80 | 60 | 15 | 170 | 145 | 32 | 58 |
| К2118Б | 300 | 200 | 120 | 150 | 100 | 80 | 60 | 15 | 170 | 135 | 32 | 50 |
| КД2120 | 360 | 240 | 150 | 180 | 115 | 80 | 60 | 15 | 195 | 162 | 30 | 60 |
| КД2122 | 420 | 280 | 180 | 210 | 140 | 110 | 80 | 20 | 220 | 190 | 40 | 60 |
| КД2124 | 500 | 340 | 210 | 250 | 170 | 130 | 100 | 20 | 280 | 225 | 40 | 60 |
| КД2126 | 600 | 400 | 250 | 300 | 200 | 130 | 100 | 20 | 350 | 285 | 50 | 70 |
| КД2128 | 710 | 480 | 300 | 360 | 240 | 170 | 140 | 30 | 370 | 310 | 50 | 75 |
| К2130 | 850 | 560 | 360 | 420 | 280 | 240 | 200 | 30 | 330 | 295 | 60 | 75 |
| КЕ2130 | 950 | 630 | 360 | 420 | 280 | 240 | 200 | 40 | 450 | 410 | 60 | 70 |
| К2132 | 1000 | 670 | 420 | 480 | 320 | 240 | 200 | 40 | 460 | 580 | 75 | 85 |
| К2232 | 1000 | 670 | 420 | 480 | 320 | 240 | 200 | 40 | 665 | 560 | 75 | 100 |
| К2234 | 1120 | 750 | 480 | 530 | 360 | - | 200 | - | 860 | 670 | 75 | 75 |
| К0134 | 1120 | 750 | 480 | 530 | 360 | 240 | 200 | 40 | 930 | 730 | 75 | 120 |

## Design characteristics of single action crank press

| Model | $A_{\Gamma}$ | $B_{\Gamma}$ | $H_{\Gamma}$ | $H_{c}$ | $N_{\Gamma \imath}$ |
| :--- | :---: | :---: | :---: | :---: | :---: |
| КД1424 | 1170 | 1200 | 2145 | 650 | 2,7 |
| КД1426 | 1270 | 1375 | 2565 | 755 | 4,7 |
| КД1428 | 1500 | 1800 | 3020 | 790 | 9,0 |
| К1430 | 1555 | 1850 | 2935 | 750 | 14,5 |
| КА1432А | 2340 | 2100 | 3465 | 820 | 19,0 |
| К2114 | 515 | 800 | 1535 | 810 | 0,4 |
| КД2114 | 600 | 825 | 1585 | 800 | 0,37 |
| К2116Б | 585 | 850 | 1715 | 830 | 0,5 |
| КД2118 | 620 | 970 | 1805 | 800 | 0,75 |
| К2118Б | 590 | 915 | 1795 | 850 | 0,8 |
| КД2120 | 965 | 1045 | 1790 | 745 | 2,0 |
| КД2122 | 990 | 1085 | 1875 | 760 | 2,0 |
| КД2124 | 1170 | 1190 | 2110 | 820 | 2,7 |
| КД2126 | 1270 | 1350 | 2420 | 835 | 4,7 |
| КД2128 | 1450 | 1730 | 2180 | 840 | 8,3 |
| К2130 | 1555 | 1850 | 2725 | 700 | 14,5 |
| К2130Б | 1440 | 1910 | 2650 | 700 | 10,0 |
| К2132 | 2100 | 2340 | 3650 | 820 | 19,0 |
| К2232 | 1790 | 2065 | 3560 | 820 | 10,0 |
| К2234 | 2300 | 2450 | 3950 | 800 | 20,0 |
| К0134 | 2640 | 2700 | 4810 | 720 | 27,0 |

Technological characteristics of simple action closed presses
One crank

| characteristic | Model |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | KA2534 | К2535А | KA2536 | К2538 | К2540 | К2542 | К2544 |  |
| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 |  |
| $P$ | 250 | 315 | 400 | 630 | 1000 | 1600 | 2500 |  |
| $h$ | 200 | 400 | 250 | 320 | 400 | 400 | 600 |  |
| $n$ | 32 | 16 | 25 | 20 | 16 | 10 | 6 |  |


| $H$ | 560 | 710 | 670 | 800 | 950 | 1120 | 1380 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $H_{n л}$ | 140 | 140 | 160 | 180 | 220 | 320 | 300 |
| $\Delta_{u}$ | 140 | 140 | 160 | 180 | 200 | 220 | 220 |
| $H_{H}$ | 560 | 560 | 460 | 750 | 960 | 1120 | 1400 |
| $L_{H}$ | 720 | 840 | 850 | 1080 | 1300 | 1620 | 1770 |
| $L_{c}$ | 860 | $!000$ | 1000 | 1250 | 1480 | 1830 | 2000 |
| $h_{\text {нв }}$ | 100 | 200 | 200 | 130 | 200 | 250 | 300 |
| $Q$ | 50 | 80 | 63 | 105 | 200 | 350 | 50 |
| $P_{\text {нв }}$ | 6,6 | 8 | 7 | 7,8 | 14 | 20 | 50 |
| $A_{o}$ | 520 | 600 | 700 | 990 | 1150 | 1560 | 1400 |
| $H_{o}$ | 560 | 530 | 500 | 620 | 750 | 980 | 1100 |
| $l_{o}^{\prime}$ | - | 150 | 130 | 140 | 200 | 150 | 300 |
| $A$ | 1570 | 5600 | 2940 | 5500 | 12000 | 20000 | 31600 |
| $A^{\prime}$ | 1000 | 1000 | 1000 | 1150 | 2460 | 5000 | 5000 |

Double crank

| Model | $P$ | $h$ | $n$ | $H$ | $H_{n \lambda}$ | $\Delta_{u}$ | $Q$ | $L_{c}$ | $h_{6 . B}$ |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| К3730А | 100 | 160 | 50 | 500 | 100 | 160 | 10 | 1450 | 40 |
| КБ3732 | 160 | 200 | 48 | 630 | 140 | 200 | 20 | 2010 | 86 |
| К3732 | 160 | 200 | $3 O$ | 600 | 130 | 160 | 32 | 2000 | 86 |
| К3534 | 250 | 200 | 25 | 630 | 160 | 250 | 28 | 2510 | 140 |
| КА3534 | 250 | 400 | 25 | 750 | 160 | 250 | 32 | 2510 | 140 |
| К3535А | 315 | 400 | 25 | 750 | 180 | 250 | 32 | 2500 | 140 |
| К3735 | 315 | 250 | 24 | 750 | 190 | 200 | 32 | 2500 | 140 |
| К3735А | 315 | 500 | 18 | 1000 | 190 | 400 | 32 | 2500 | 140 |
| К3536 | 400 | 250 | 20 | 750 | 190 | 200 | 80 | 2500 | 180 |
| К3537 | 500 | 250 | 20 | 750 | 220 | 200 | 80 | 2500 | 180 |
| КБ3537 | 500 | 250 | 25 | 750 | 190 | 200 | 80 | 2500 | 180 |
| К3539 | 800 | 315 | 17 | 900 | 250 | 250 | 160 | 2500 | 180 |
| КА3539 | 800 | 315 | 17 | 900 | 250 | 250 | 80 | 2500 | 180 |
| К3540 | 1000 | 400 | 20 | 1060 | 250 | 320 | 26,6 | 3150 | 150 |
| К3542 | 1000 | 400 | 16 | 1060 | 300 | 360 | 320 | 5000 | 150 |
| К3544 | 2500 | 500 | 11 | 1250 | 300 | 500 | 500 | 4000 | 150 |
| К3546 | 4000 | 630 | 10 | 1500 | 320 | 200 | 800 | 5000 | 250 |
| К3046 | 4000 | 500 | 10 | 1180 | 320 | 300 | 800 | 12000 | 200 |

## Dimensions of places for stamp installing and design characteristics of closed simple action crank presses

| Model | $A_{c}$ | $B_{c}$ | $A_{n 3}$ | $B_{n 3}$ | $A_{\Gamma}$ | $B_{\Gamma}$ | $H_{\Gamma}$ | $N_{\text {г. }}$ |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| КА2534 | 850 | 850 | 670 | 650 | 3000 | 2970 | 5140 | 27 |
| К2535А | 1000 | 1000 | 800 | 800 | 3100 | 3140 | 5770 | 39,9 |
| КА2536 | 1000 | 1000 | 800 | 800 | 3410 | 3140 | 5930 | 39,9 |
| К2538 | 1250 | 1250 | 1000 | 1000 | 3280 | 3470 | 6190 | 58 |
| К2540 | 1250 | 1250 | 1250 | 1000 | 4540 | 3800 | 6990 | 77 |
| К2542 | 1800 | 1800 | 1440 | 1500 | 5050 | 4620 | 8720 | 125 |
| К2544 | 2000 | 2800 | 1600 | 2100 | 5390 | 4870 | 9125 | 132 |
| К3730А | 1250 | 800 | 1400 | 700 | 3085 | 1740 | 4010 | 17 |
| КБ3732 | 2000 | 1250 | 1950 | 1000 | 3550 | 2250 | 4570 | 20 |
| К3732 | 2000 | 1250 | 1950 | 1000 | 3125 | 2400 | 4490 | 13 |
| К3735А | 2470 | 1250 | 2440 | 1000 | 4290 | 2210 | 5890 | 40 |
| КА3534 | 2500 | 1250 | 2470 | 1000 | 4070 | 2700 | 5840 | 29 |
| К3534 | 2500 | 1250 | 2470 | 1000 | 4480 | 2900 | 5085 | 30 |
| К3535А | 2500 | 1250 | 2470 | 1000 | 4555 | 2900 | 5855 | 50 |
| К3536 | 2500 | 1400 | 2400 | 1000 | 4350 | 2470 | 5660 | 40 |
| К3735 | 2500 | 1250 | 2440 | 1000 | 4290 | 2210 | 5385 | 40 |
| К3537 | 2500 | 1400 | 2440 | 1200 | 4350 | 2470 | 5 b60 | 40 |
| КБ3537 | 2500 | 1400 | 2400 | 1200 | 4265 | 2870 | 6220 | 45 |
| К3539 | 2500 | 1500 | 2280 | 1300 | 4485 | 2800 | 6245 | 55 |
| КА3539 | 2500 | 1500 | 2470 | 1320 | 4375 | 2975 | 6520 | 58 |
| К3540 | 3150 | 1800 | 3066 | 1700 | 6000 | 4740 | 7970 | 160 |
| К3542 | 5000 | 2000 | 4750 | 1800 | 6500 | 4800 | 8450 | 160 |
| К3544 | 4000 | 2000 | 3750 | 1800 | 7980 | 5000 | 9015 | 160 |
| К3546 | 5000 | 2000 | 5000 | 2000 | 7640 | 5980 | 1490 | 250 |
| К3046 | 11500 | 2000 | 11000 | 1900 | 12570 | 6750 | 9850 | 250 |
| К4540 | 4620 | 2464 | 4570 | 2440 | 7970 | 5440 | 9970 | 160 |
| К4542 | 3150 | 2500 | 3100 | 2500 | 6240 | 5420 | 7000 | 155 |
| К4546 | 5000 | 3000 | 4950 | 3000 | 8520 | 7000 | 9125 | 245 |

ДЛЯ НОТАТКІВ

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## МЕТОДИЧНІ ВКАЗІВКИ

# до виконання курсової роботи з дисципліни «ТЕХНОЛОГІЯ ОБРОБКИ ДЕТАЛЕЙ ТИСКОМ» 

Для здобувачів вищої освіти<br>з числа іноземних громадян за спеціальністю 131 - «Прикладна механіка»

Комп’ютерне макетування та верстка А.П. Катрич

Формат 60x90/16. Обл. вид. арк. 1,43. Тираж 10 прим. Зам. № 3097.

Тернопільський національний технічний університет імені Івана Пулюя.
46001, м. Тернопіль, вул. Руська, 56.
Свідоцтво суб'єкта видавничої справи ДК № 4226 від 08.12.11.


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